# m RI 9123 

# LIBRARY <br> SPOKANE RESEARCH CENTER RECEIVED 

NOV 51987

U.S. BUREAU OF M:NFS<br>E. 315 MONTG AAR AVE. SPOKANE, WA 99207

# Large-Scale Testing of the Ripper Fragmentation System 

By David A. Larson, Roger J. Morrell, and David E. Swanson

## Report of Investigations 9123

## Large-Scale Testing of the Ripper Fragmentation System

By David A. Larson, Roger J. Morrell, and David E. Swanson

## Library of Congress Cataloging in Publication Data:

## Larson, David A.

Large-scale testing of the ripper fragmentation system.
(Bureau of Mines report of investigations; 9123)
Bibliography: p. 16.
Supt. of Docs. no.: I 28.23: 9123.

1. Drag bits (Drilling and boring)-Testing. 2. Rock mechanics. 3. Quarries and quarrying. I. Morrell, Roger J. II. Swanson, David E. III. Title. IV. Series: Report of investigations (United States. Bureau of Mines); 9123.

TN23.U43 [TN281] 622 s [622',23] 87-600010
Abstract ..... 1
Introduction ..... 2
Background ..... 2
Previous Bureau work ..... 2
The need for full-scale tests. ..... 3
Test equipment ..... 3
Large-scale ripper tester. ..... 3
Drag cutters ..... 5
Concrete test specimens ..... 5
Data acquisition ..... 5
Test design. ..... 6
Experimental results and analysis ..... 7
Bit forces versus cutting depth ..... 7
Bit forces versus cutting width. ..... 8
Bit forces versus position of bit on face. ..... 9
Specific energy of cutting. ..... 10
Bit wear ..... 11
Dust generation. ..... 12
Muck size ..... 12
Application of results ..... 14
Productivity calculations ..... 14
Cutter force and power calculations ..... 14
Summary and conclusions ..... 15
References ..... 16
Appendix.--Raw data. ..... 17
ILLUSTRATIONS

1. Ripper cutting method. ..... 3
2. Large-scale ripper tester ..... 4
3. Cutting action of ripper tester showing push-pull cylinders ..... 5
4. Nine-in wide ripper bits tested at $0^{\circ}$ and $10^{\circ}$ rake angles ..... 6
5. Average cutting force as a function of cutting depth for a 4-in cutting width. ..... 7
6. Average cutting force as a function of width for a $1-1 / 2-i n$ cutting depth.. ..... 8
7. Bit cutting force as a function of horizontal position. ..... 9
8. Specific energy as a function of the cross-sectional area of the cut ..... 10
9. Size fractions of typical muck from full-face cut. ..... 13
TABLES
10. Ripper tester specifications ..... 4
11. High-strength $\left(9,200-1 b f / i n^{2}\right)$ concrete mix design. ..... 5
12. Test design variables ..... 6
13. Maximum drag bit life ..... 12
14. Sieve analysis of ripper cuttings ..... 12

|  | UNIT OF MEASURE ABBR | USED IN | REPORT |
| :---: | :---: | :---: | :---: |
| ft | foot | in/s | inch per second |
| $\mathrm{ft} / \mathrm{h}$ | foot per hour | 1b | pound |
| $f t \cdot 1 \mathrm{bf}$ | foot pound (force) | 1bf | pound (force) |
| $\mathrm{ft} / \mathrm{s}$ | foot per second | 1bf.in | pound (force) inch |
| g | gram | $1 \mathrm{bf} / \mathrm{in}^{2}$ | pound (force) per square inch |
| gal | gallon |  |  |
| gal/min | gallon per minute | min | minute |
|  | hour | $\mathrm{min} / \mathrm{h}$ | minute per hour |
| h |  | pct | percent |
| hp | horsepower |  |  |
|  |  | s | second |
| Hz | hertz |  |  |
|  |  | st | short ton |
| in | inch |  |  |
| in ${ }^{2}$ | square inch | wt pct | weight percent |
| $1 n^{3}$ | cubic inch |  |  |
| in. $1 \mathrm{bF} / \mathrm{in}^{3}$ | inch pound (force) per cubic inch |  |  |

# LARGE-SCALE TESTING OF THE RIPPER FRAGMENTATION SYSTEM 

By David A. Larson,' Roger J. Morrell, ${ }^{2}$ and David E. Swanson'


#### Abstract

This is the second in a series of Bureau of Mines reports which describe experiments designed to devise an efficient, economic mechanical fragmentation technique for hard rock. The previous study described the development of the ripper cutting technique to successfully cut hard rock with 3 - to 6 -in-wide drag cutters. This study was conducted to test the ripper cutting technique at full scale using 9 -in-wide drag cutters. The experiments were conducted with a special test device that cut shallow, $6-$ by $6-\mathrm{ft}$ openings in $9,200-1 \mathrm{bf} / \mathrm{in}^{2}$ compressive-strength concrete blocks.

The tests showed that ripper cutting can excavate large-scale openings of the type required by the mining industry at a production rate that outperforms the conventional drill-blast technique. Moreover, the system can produce openings of various sizes and shapes, is simple to operate and maintain, and produces very little dust or noise during operation. The system appears to meet all of the criteria for a successful hard-rock mining machine.


[^0]
## BACKGROUND

The Bureau of Mines has long recognized the benefits of mechanized excavation for hard-rock underground mines and has initiated a program to conceive a simple but efficient way to fragment hard rock. It is anticipated that such a fragmentation technique will be used to extend the application of mechanical mining machines into hard rock.

The application of a high-production, low-cost fragmentation technique could revolutionize hard rock mining in the same way as the continuous miner revolutionized underground coal mining, and there have been many attempts to produce such a machine. The problem has always been the lack of a suitable fragmentation technique around which to construct the machine. Analysis of the needs of the underground mining industry has shown that there are several essential characteristics for a successful fragmentation system (4). ${ }^{3}$ These characteristics are as follows:

1. The system must be able to cut a wide range of rock types under a variety of underground environments. The system must be able to cut rocks from soft shales through hard granites and basalts. It must be able to operate in massive rock, in broken and jointed rock, in mixed face conditions, and in high water inflows. Essentially, the technique must be versatile enough to handle any conditions that could arise under normal mining operations.
2. The system must be economical. Besides being able to fragment the rock, it has to do so at the same or at a lower cost than conventional techniques. Personnel requirements for maintenance and operation must be minimal. Therefore, the equipment must be amenable to automatic operation, and it must be simple, rugged, and inexpensive.

[^1]3. The system must be able to achieve a high rate of production and a consistent product size. It must be able to achieve 500 to 700 st per shift, yield a controlled product size, and be able to operate in conjunction with continuous haulage systems.
4. The system must be capable of operating in current mining operations. That is, it must be able to operate in a mining operation without requiring extensive changes in the basic mining plan. Therefore, the system must be able to excavate openings of required size and shape, and it must be maneuverable and mobile.
5. The system must operate without creating hazards such as dust, noxious gases, radiation, large amounts of heat, etc. The system must be safer and better from a health standpoint than conventional methods.

## PREVIOUS BUREAU WORK

Earlier Bureau investigations (등) identified a method that promises to meet the above-listed requirements for a fragmentation system. This method is called ripper cutting and is illustrated in figure 1.

The forces acting on the bit in the ripper cutting system are defined in the conventional manner, with $\mathrm{F}_{\mathrm{C}}$ representing the cutting force and $F_{n}$ the normal force. The system uses a single large drag cutter to cut the rock in a series of parallel circular cuts. The system uses the previous cut as one free face, and the depth of cut is limited to onehalf or less of the cut width. Using this technique, the Bureau previously conducted experiments in four rocks ranging in strength from 10,000 to $27,000 \mathrm{lbf} / \mathrm{in}^{2}$ (5). Drag bits, 3- and $6-i n$ wide, were used to make cuts as deep as 2 -in per pass. The results of these tests were very encouraging; the energy efficiency of the method was from 50 to 200 pet better than for tunnel boring machines, and 12 to 58 pct better than for roadheaders (5). Moreover, the process can be scaled up to achieve a high


FIGURE 1.-Ripper cutting method.
production rate, and the dust levels and cutter wear appeared to be very low.

Dust levels and cutter wear were not actually measured in these earlier tests, but the following general observations
were made: Essentially, no cutter wear was observed during any of the tests except for a slight polishing of the cutting edge, and no dust cloud was visually observed around the bit during cutting (5).

## THE NEED FOR FULL-SCALE TESTS

Based on the previous test results using 3- and 6-in-wide drag bits, a decision was made to study ripper cutting at full scale. Full-scale testing was considered necessary because the small-scale tests did not model the complete circular cut, and it was belleved that as the excavation deepened, the cutting process could be adversely affected. Full wscale cutting was also considered necessary in order to generate the accurate cutting force and production rate data that would be necessary to judge the usefulness of the system. Full-scale ripper cutting is considered to include openings with cross sections of at least 6 by 6 ft and bit widths of 9 in or more. The maximum opening size and maximum bit size has not yet been defined for ripper cutting. The remainder of this report describes the Bureau's efforts to design and test ripper cutting at full scale.

## TEST EQUIPMENT

## LARGE-SCALE RIPPER TESTER

For the large-scale excavation tests of ripper cutting, it was determined that the bit had to be at least 9 in wide and the opening to be cut had to be at least 6 by 6 ft to minimize scaling effects. This required the construction of a special test device that could power a bit of this size and a test sample large enough to accommodate a 6 - by 6 -ft opening (the test cut). The device developed for these tests is called the large-scale ripper tester. It uses two push-pull hydraulic cylinders to power the bit through a $185^{\circ}$ vertical cutting arc and generates $150,000 \mathrm{lbf}$ of cutting force at the bit at $3,000 \mathrm{lbf} / \mathrm{in}^{2}$ hydraulic pressure. It also has a rotation system to move the bit across the face between
cuts and an advance system to feed the machine forward. The essential features of the ripper tester are shown in figure 2 , and its performance specifications are shown in table 1.

The operation of the ripper tester is as follows: First, the machine is advanced forward by the thrust cylinders the increment to be cut. This is usually 1 to 4 in or from one-third to one-half the cut width. The advance system is then locked and not moved forward again until the entire width of the face has been cut by the bit. The machine is rotated horizontally to begin the first cut on the face. This cut can be either at the center of the face or on one side of the face. When the bit is in the proper position, the machine is locked in place by four vertical jacks. This


FIGURE 2.-Large-scale ripper tester.

TABLE 1. - Ripper tester specifications

| Component | Specifications |
| :---: | :---: |
| Cutting system.. | 2 8-in-diam-bore hydraulic cylinders, $150,0001 \mathrm{bf} \max$ at 3,000 $1 \mathrm{bf} / \mathrm{in}^{2}$; speed, $+3-1 / 2 \mathrm{in} / \mathrm{s}$ at $30 \mathrm{ga} 1 / \mathrm{min}$. |
| Thrust system.... | 2 6-in-diam-bore hydraulic cylinders, 600,000 lbf max at $10,0001 \mathrm{bf} / \mathrm{in}^{2}$. |
| Rotation system.. | 2 6-in-diam-bore hydraulic cylinders, 600,000 lbf max at $10,000 \mathrm{lbf} / \mathrm{in}^{2}$. |
| Holddown system.. | 4 100-st jacks, 2 forward, 2 rear. |
| Cutter........... | ```9- to 12-in wide crown type, bolted on, heat-treated tool steel.``` |
| Dust suppression. | Water spray. |

prevents the machine from moving while cutting. When the machine is in proper position and locked in place, the cutting system is activated. The push-pull cutting cylinders swing the bit from the bottom to the top of the face through an arc of about $185^{\circ}$. This operation is shown in figure 3. As the bit swings through this are, the rock cuttings are collected in the cutterhead and dropped onto a muck chute at the end of the cut after the bit has returned to its original bottom position. After completion of the cut, the holddown jacks are released, the machine is rotated horizontally to the next cut position, the jacks are reset, and the machine is ready for another cut. Each succeeding cut used the edge of the previous cut as a free face, and cuts can be less than the full width of the bit. Each cut requires about 22 s to complete, and repositioning between cuts requires about 30 s .


FIGURE 3.-Cutting action of ripper tester showing push-pull cylinders.

## DRAG CUTTERS

The cutters used in these experiments were specially designed drag cutters constructed of tool steel and heat treated to a hardness of 58 Rockwell C. The bits were 9 in wide, 1 in thick, and had a circular-shaped cutting edge. The bits were made with a $0^{\circ}$ and a $10^{\circ}$ rake angle, and the clearance angel was held at $10^{\circ}$ for all bits. The bits were attached to the bit holder with three bolts, and bit changing took one operator about 5 min . The bits are shown in figure 4.

## CONCRETE TEST SPECIMENS

All of the large-scale excavation tests were conducted in a single 3 - by 8 - by $8-\mathrm{ft}$ block of concrete. The concrete was poured into a steel form, which was constructed as part of the test device. The concrete block was solid and massive with no visible flaws or weaknesses. The concrete had a 28 -day compressive strength of $9,200 \mathrm{lbf} / \mathrm{in}^{2}$. It was made with 3/4-in limestone aggregate, which was considered to be representative of a nonabrasive low-strength rock. The formulation for a cubic yard of the highstrength concrete is given in table 2.

## DATA ACQUISITION

During each test, the cutting force and normal force on the bit were measured and recorded. This was accomplished by measuring the hydraulic pressure in the cutting circuit and the thrust circuit.

TABLE 2. - High-strength (9, 200-1bf/in ${ }^{2}$ ) concrete mix design

Ingredient 1b

Portland type 1 cement............... 971
Water................................... 298
Sand........................................ 1, 049
Aggregate, $3 / 4$ in angular 11 mestone 1,643


FIGURE 4.-Nine-inch-wide ripper bits tested at $0^{\circ}$ and $10^{\circ}$ rake angles.

The pressure in each circuit times the cross-sectional area of the cylinder, times the appropriate mechanical linkage geometry, yielded the forces on the bit. The pressures were measured with straingauge pressure transducers and recorded on strip chart recorders. The cutting force was considered accurate within 10 pct, as were the specific energies and rock numbers, which were computed from the cutting force data. The normal bit forces were erratic during calibration, due to varying frictional forces, and can only be considered to be rough estimates of the real values. Therefore, throughout this report, the normal forces are given only as a percentage of the cutting force and are not plotted separetly. The normal force estimates are still considered extremely important to the design of ripper systems, and the estimates given for each value can still be used as the best-case and worst-case design data.

TEST DESIGN
The experimental test program had two general goals: first, to determine if ripper cutting could excavate a full-size opening in a massive rock specimen, and second, to generate sufficient engineering data on the method to allow the calculation of realistic production rates and energy consumption. To accomplish these goals, a test plan was developed, using results from the previous smallscale tests to set the initial operating conditions. The independent variables that were tested are shown in table 3.

TABLE 3. - Test design variables
Variable Values used

Bit rake angle................ $0^{\circ},+10^{\circ}$
Cutting depth............in.. 1, $1-1 / 2,2$
Cutting width............in.. 2, 4, 6, 9

The cutter type used throughout the test program was a 9 -in-wide crown-type drag bit with a $10^{\circ}$ clearance angle. The cutting widths studies were $2,4,6$, and 9 in. The cutting speed was held constant at $3 \mathrm{in} / \mathrm{s}$, and all cuts were made from bottom to top. While $3 \mathrm{in} / \mathrm{s}$ is considerably slower than the $12-\mathrm{in} / \mathrm{s}$ bit speed that would be used in field cutting, it was not expected to have any
effect on the bit forces reported in this paper, based on previous work (2). The dependent variables that were measured or calculated were bit cutting force in pounds (force), bit normal force, in pounds (force), and specific energy, in inch pounds (force) per cubic inch. A total of 82 tests was conducted. The raw data are shown in appendix A.

EXPERIMENTAL RESULTS AND ANALYSIS

The depth of cut varies over the length of the cut. The cut depth starts at zero and gradually increases to a maximum at the center and then gradually decreases to zero as the cut is completed (fig. 3). Therefore, the depths and the areas of the cuts referred to in this report are calculated at the center of the cut, where the depth and cross-sectional area are greatest.

When using the ripper fragmentation method, the width of cut is independent of the size of the bit. The width of the cut is defined as the width of the cut made by the bit and can be varied simply by rotating the machine to a greater or lesser extent between cuts. A 9-in-wide bit can be used to make cuts from 0 to 9 in wide, depending upon the rotation of the machine.

## BIT FORCES VERSUS CUTTING DEPTH

The average cutting force on the bit was plotted as a function of the maximum cutting depth for both bits tested, at a constant width of 4 in (fig. 5). The best-fit equations for these curves are

$$
\mathrm{F}_{\mathrm{c}}=10,439 \mathrm{D}^{0.52}
$$

for the $0^{\circ}$ rake angle bit
and

$$
\mathrm{F}_{\mathrm{c}}=10,297 \mathrm{D}^{0.52}
$$

for the $10^{\circ}$ rake angle bit,
where $F_{c}=$ average cutting force, lbf,
and $\quad D=$ cutting depth, in.

These equations, with exponents less than 1.0 , show that the cutting force increases at a slower rate than the depth of cut. This indicates that the cutting process becomes more efficient as the depth of cut increases. The intuitive explanation for this improvement is that deeper cuts produce more large chips and fewer small chips. Since a smaller new fracture surface is created in making large chips, less total fracture surface energy is required, and the process becomes more efficient.

The curves also show that the $0^{\circ}$ bit and the $10^{\circ}$ bit performed about equally well, so there is no basis for selecting one over the other. This is in contrast


FIGURE 5.-Average cutting force as a function of cutting depth for a 4 -in cutting width.
to previous work (5) that showed the bit rake angle to have a significant effect on cutting force. Future tests will be conducted in a wider range of rock samples to clarify this relationship.

While these experiments on cutting depth were conducted using only a 4-inwidth of cut, the general result should be valid for all other cut widths as well. That is, the cutting force will increase with an increasing depth of cut. The actual rate of increase, however, may be slightly different than that obtained for the 4 -in-wide cut.

All of the previous discussion has dealt with average cutting forces. However, the cutting force acting on the bit during cutting is really a series of force highs and lows with a major frequency of approximately $1 / 2$ to 7 Hz . To fully characterize the cutting forces acting on the bit, the peak cutting force should also be analyzed. The peak cutting force is the force that would be used to design the structural integrity of a mining machine. In this investigation, the peak cutting force was defined as the average of the three highest forces recorded during a cutting run. This averaging was necessary to achieve repeatability between similar experimental runs. The peak cutting force was found to be a function of the average cutting force as follows:

$$
F_{c p}=2.5 F_{c},
$$

where $F_{c p}=$ peak cutting force, lbf,
and $\quad F_{c}=$ average cutting force, 1 bf .
The normal forces were also measured during these experiments, but, as noted earlier, the normal force data are considered to be estimates only and, therefore, were not plotted along with the cutting force. The data show, however, that on the average, the normal force as a function of cutting force was

$$
F_{n}=1.2 F_{c}
$$

where both $\mathrm{F}_{\mathrm{n}}$ and Fc are average values.

BIT FORCES VERSUS CUTTING WIDTH
Figure 6 shows the average cutting force as a function of cutting width for both bits tested. The best-fit equations for these curves are

$$
\mathrm{F}_{\mathrm{c}}=6,204 \mathrm{~W}^{0.48}
$$

for the $0^{\circ}$ rake angle bit
and

$$
\mathrm{F}_{\mathrm{c}}=8,046 \mathrm{w}^{0.26}
$$

for the $10^{\circ}$ rake angle bit,
where $\quad W=$ width of cut, in.
These curves and equations show the influence of cutting width and bit rake angle on cutting force. These equations are very similar to the depth-force relationship shown in the previous section in that the cutting force increases at a slower rate than the cutting width, and the cutting process becomes more efficient as width is increased.

The graphs also show that the $10^{\circ}$ rake angle bit required slightly less cutting force than the $0^{\circ}$ bit, but only for the


FIGURE 6.-Average cutting force as a function of width for a $11 / 2$-In cutting depth.
widest cuts. For the smaller cuts, there was no appreciable difference in cutting force. Again, this result is surprising, based on previous work that showed the $10^{\circ}$ rake angle bits to be more efficient than the $0^{\circ}$ bits (5). It is expected that more testing in a wider variety of rocks will show the effects of bit rake angle.

BIT FORCES VERSUS POSITION OF BIT ON FACE

The ripper fragmentation method makes a series of vertical circular cuts across the entire face of the rock being excavated. Because of the geometry of the excavation produced, the right
and left halves of the excavation are mirror images of each other, so that corresponding cuts to the right and left of center are identical and should experience the same cutting forces. The geometry of each cut varies, however, as the cut proceeds from the bottom to the top of the excavation.

Figure 7 shows the geometry of the various cuts across the face and an example of the average cutting forces associated with each cut. The center cut is usually the first cut made on the face and has a high cutting force because of the full width of the cut and the lack of a free face to the side. The remaining cuts, starting next to the center cut and



FIGURE 7.-Bit cutting force as a function of horizontal positlon. (The sequence of cuts 1 through 14 corresponds to actual cuts 69 through 82 as shown In the appendlx.)
moving to first to one side of the excavation and then the other, should have gradually reduced forces as the bit approaches the sides. However, since the excavation is quite narrow, the cut volumes are nearly the same and, hence, the average force for each cut is approximately equal. The last cut on each side, which defines the right or left boundary of the excavation, requires very high forces. This is because these corner cuts are much more confined than any of the other cuts. Cuts of the same width and depth on the right side of the excavation require approximately the same cutting force as cuts of equal size on the left side. Thus, experimental data for one side of the face cinn be sompared directly to data for the other side.

## SPECIFIC ENERGY OF CUTTING

The specific energy of an excavation process is defined as the energy required to fragment a given volume of rock. The units used are inch pounds (force) per cubic inch, and the lower the value, the more efficient the process is. The mathematical definition of specific energy is as follows:

$$
E_{s}=E / V
$$

where $E_{S}=$ specific energy, in•lbf/in ${ }^{3}$,

$$
\begin{aligned}
\mathrm{E}= & \text { total energy consumed during } \\
& \text { cutting, } 1 \mathrm{bf} \cdot \mathrm{in} .
\end{aligned}
$$

and $\quad V=$ total volume of rock cut, in ${ }^{3}$.

The specific energy of cutting is affected by many factors, including the type of material being cut, the type of bit used, and the geometry of the cut. Figure 8 shows how the specific energy varies as a function of bit type and the cross-sectional area of the cut.

For both bits tested, the specific energy decreased steadily as the area of


FIGURE 8.-Specific energy as a function of the crosssectional area of the cut.
the cut increased (fig. 8). The explanation for this phenomenon is that deeper cuts produce a larger proportion of large chips, so the overall surface area created per unit volume is less than it would be for shallower cuts. This trend was observed for all the tests performed, and it is expected that it would also be observed for larger cuts, but this is not known at this time.

The data showed no appreciable difference in specific energy between the $0^{\circ}$ and $10^{\circ}$ bits. As noted earlier, this is in contrast to previous work (5) which showed a definite effect of rake angle. Future tests in a wider variety of rocks should clarify the relationship of rake angle and specific energy.

While the specific energy of cutting defines the efficiency of the cutting process, it is necessary to take into account the hardness of the rock being cut if the overall efficiency of a given
method is to be compared to other cutting methods. To allow comparison of different cutting methods in different rocks, the following method of measuring cutting efficiency, as described by Hughes (3) and Gaye (1), was used:

The efficiency parameter, or rock number, $N_{R}$, is defined as follows:

$$
N_{R}=\frac{\sigma c}{E_{s}}
$$

where $N_{R}$ is dimensionless,

$$
\begin{aligned}
\sigma c= & \text { unconfined compressive } \\
& \text { strength of rock being } \\
& \text { cut, } 1 \mathrm{bf} / \mathrm{in}^{2},
\end{aligned}
$$

and $\quad E_{s}=$ specific energy of the process, in•lbf/in ${ }^{3}$.
$E_{S}$ is calculated as the energy consumed, in pound (force) inches, divided by the volume of the cuttings produced, in cubic inches, which yields an $E_{s}$ value in inch pounds (force) per cubic inch. $E_{s}$ in inch pounds (force) per cubic inch reduces to pounds (force) per square inch, so $N_{R}$ becomes a dimensionless number. The larger the rock number $N_{R}$, the more efficient the process is.

The $\mathrm{N}_{\mathrm{R}}$ calculated at the most efficient operating point for the ripper tester was 6.2. This was achieved using the $10^{\circ}$ bit with a $13.5-$ in $^{2}$ cut. For comparison, $N_{R}$ values given by Gaye (1) are 4 to 6 for tunnel boring machines with disk cutters and 8 for roadhead excavators. Thus, the ripper cutting technique is slightly more energy efficient than tunnel boring machines and somewhat less efficient than roadheaders. However, the efficiency number $N_{R}$ is also dependent on the cutter geometry. Previous work (5) with the ripper in natural rock yielde $\bar{d} N_{R}$ values of up to 12. Thus, it is very likely that further improvements can be made in
the cutting process with the proper selection of cutter geometry.

## BIT WEAR

The two bits used in these experiments formed a wear flat about 0.25 -in wide on the clearance side of each bit after traveling about 500 ft . This amount of wear did not seriously affect the cutting action, and both bits were used throughout the experiments without resharpening.

It would be advantageous to be able to predict bit life, as bit cost has a significant effect on the economics of the fragmentation system. However, the wear experienced in these tests is difficult to directly compare to field cutting in rock because of significantly different conditions. For example, it is well known that the wear experienced by a bit is directed related to the forces on the bit, the bit speed and metallurgy, and the hardness and abrasiveness of the material being cut.

A 1978 study done under Bureau contract S3371323 provided accurate wear-1ife data for drag bits in several rock types. These bit-wear data should be directly applicable to ripper drag bits, since the bit speed used was the same as predicted for full-scale ripper cutting and the cutting geometry was identical to fullscale ripper cutting geometry. The results, which should directly predict the wear life of full-scale tungsten carbide ripper bits, are shown in table 4.
The best bit lives were achieved at the lowest bit speed of $12 \mathrm{in} / \mathrm{s}$, when adjacent cuts overlapped each other and when a water mist was used to cool the bit. Bit lives similar to those shown in table 4 could be expected for ripper bits in similar rocks. It is estimated that for a full-size ripper with a 24 -in-wide bit, the bit would travel about $9,800 \mathrm{ft}$ in an

TABLE 4. - Maximum drag bit life, ${ }^{1} \mathrm{ft}$

| Bit rake angle | $\begin{gathered} \text { Saridstone } \\ \left(9,000 \mathrm{lbf} / \mathrm{in}^{2}\right) \end{gathered}$ | $\begin{gathered} \text { Limestone } \\ \left(9,000 \mathrm{lbf} / \mathrm{in}^{2}\right) \end{gathered}$ |
| :---: | :---: | :---: |
| $+15^{\circ} \ldots$ | 500 | 1,200 |
| $0^{\circ}$. $\ldots .$. | 300 | 2,500 |
| -15 ${ }^{\circ}$... | 1,100 | 10,400 |
| $\begin{aligned} & \text { Trona } \\ & 40,000 \mathrm{ft} \end{aligned}$ | , $000 \mathrm{lbf} / \mathrm{in}^{2}$ ): <br> t all 3 angles | no wear after |

8-h shift. Thus, in some rocks, no bit changes would be required over the entire shift; while in harder, more abrasive rocks, as many as 10 bit changes would be required. Since bit changes can be accomplished by one person in 5 min , and 5 to 10 resharpenings should be achievable for each bit, bit wear and bit changing should not be a serious problem in most nonabrasive rocks. However, bit wear cannot be accurately predicted until more bit-wear studies have been conducted.

## DUSTi GENERATION

During the cutting process, a small amount of airborne dust was created. To control this dust, a single water-spray nozzle was mounted on the cutting head such that the leading edge of the bit was engulfed in the water spray. The nozzle used was a hollow-core spray nozzle with a $0.05-\mathrm{in}$-diam orifice, and it used about 0,1 gal of water per minute. This single spray was judged, by visual observation, to be very effective in knocking down the dust at the bit. The large dust cloud
produced during dry cutting was nearly eliminated when the single water spray was used. For even greater effectiveness, a second spray nozzle should be mounted to cover the area directly behind the trailing edge of the bit, since a substantial amount of dust was also produced in this area. Based on these results, it appears that several water sprays surrounding the bit should effectively control the dust generated during ripper cutting.

## mUCK SIZE

It was observed that the largest rock chip formed was approximately square in cross section, with maximum dimensions equal to the width of the cut. The thickness was equal to or less than the depth of the cut. A 9-in-wide cut, 2 in deep, would therefore produce a maximum size chip approximately 2 by 9 by 9 in. The screen size of tunnel boring machine muck is predominantly ( 20 to 30 pct) about 1 to 2 in, with the largest pieces being about 1 by 3 by 9 in . The sieve analysis of a full-face cut 2 in deep and 6 in wide is shown in table 5, and the cuttings are shown in figure 9. Note the dominance of the large sizes and the lack of smaller particles. The greater proportion of larger size pieces, in comparison with cuttings produced by conventional mechanical fragmentation techniques, reduces the amount of surface area created, which in turn reduces the specific energy of cutting.

TABLE 5. - Sieve analysis of ripper cuttings

| Size fraction as shown in figure 9 | Screen size ${ }^{1}$ | $\begin{gathered} \text { Weight, } \\ g \end{gathered}$ | Wt pct | Cumulative wt pct |
| :---: | :---: | :---: | :---: | :---: |
| A...... | - $8+2$ in. | 6,657 | 42 | 42 |
| $B$ | $-2+1 / 2$ in. | 4,716 | 30 | 72 |
| $C$. | -1/2 in + 4 mesh. | 2,295 | 14 | 86 |
| D. | $-4+10$ mesh | 908 | 6 | 92 |
| $E$. | $-10+20$ mesh | 627 | 4 | 96 |
| $\underline{F}$. | -20 mesh....... | 614 | 4 | 100 |

[^2]

FIGURE 9.-Size fractions of typical muck from full-face cut using $10^{\circ}$ bit at a 2 - in depth and 6 -in width of cut. (See table 5 for screen sizes of fractions shown.)

## APPLICATION OF RESULTS

This section illustrates the potential use of the ripper fragmentation system. The ripper fragmentation system is designed to be an alternative to the drillblast method of fragmentation in most sedimentary rocks with compressive strengths up to $27,000 \mathrm{lbf} / \mathrm{in}^{2}$. The system could be used either to bulk mine or to drive development openings in sizes from approximately 6 by 6 ft to 20 by 20 ft or more.

To illustrate the use of the ripper cutting method in a mining environment, the following mining scenario was developed: A mining machine using the ripper cutting method is used to drive 10 - by 10-ft rectangular headings in a massive nonabrasive limestone with an unconfined compressive strength of $18,0001 \mathrm{bf} / \mathrm{in}^{2}$. The size of the opening and the rock type were selected to show the most likely areas for application of the method. The 10- by $10-\mathrm{ft}$ stope size is a scale-up from the 6 - by $6-f t$ openings produced in the laboratory, but is considered reasonable, given the simple construction of the ripper machine. The $18,000-1 b f / i n^{2}$ limestone was chosen to represent a common mining environment and is well within the cutting capabilities of the system, which has successfully cut $27,000-1$ bf/in ${ }^{2}$ rock in previous experiments.

Once the heading size and rock type are specified, the next step in analyzing a ripper cutting system is to determine the size of the single ripper cutter to be used. The bit size is related to the size of the heading to be excavated, the production rate to be achieved, the cutting forces acting on the bit, and the total system power requirements. In selecting the bit size, all of these factors should be balanced. The laboratory work indicated that a reasonable bit size would be about one-tenth of the width of the heading to be excavated. Thus, for the $10-\mathrm{ft}$ wide heading chosen for this analysis, a bit width of about 1 ft is indicated. Once the bit width is selected, the depth of cut can be specified. Previous work with 3- and 6-in bits showed that efficient cutting was achieved at cutting depths that were from
one-third to one-half of the bit width. Therefore, for a l2-in-wide bit, the cutting depth should be between 4 and 6 in. Finally, the bit speed should not exceed $1 \mathrm{ft} / \mathrm{s}$ in most rock types, if cutter wear is to be kept to reasonable rates. Given these initial conditions, the productivity, bit forces, and total power requirements of the ripper cutting method can be calculated.

## PRODUCTIVITY CALCULATIONS

In the calculations that follow, all work is done in a $10-$ by $10-f t$ heading in $18,000-1 b f /$ in $^{2}$ rock. The bit is 12 in wide and takes a 5-in depth of cut. The cutting path is circular and is $15-\mathrm{ft}$ in length. The cutter speed is set at 12 in/s to keep heat buildup and cutter wear under control.

1. Since each cutter swing is 15 ft long and the bit moves at $1-\mathrm{ft} / \mathrm{s}$, each cutter swing requires 15 s to complete. It is estimated that 10 s will be required to return and reposition the bit for the next cut. Hence, each cutter swing requires 25 s.
2. The 12 -in-wide cutter covers the 10-ft-wide face in 10 cuts, each requiring 25 s for a total of 4.16 min . One complete cycle across the face advances the heading 5 in. This yields an instantaneous rate of $6 \mathrm{ft} / \mathrm{h}$.
3. A total of $10 \mathrm{~min} / \mathrm{h}$ is required to move the machine ahead and reset the jacks, so the actual production rate is $5 \mathrm{ft} / \mathrm{h}$.
4. Finally, in a 8-h shift, assume 2 h are lost due to delays caused by ventilation, support, cutter changes, and scheduled and unscheduled maintenance. Thus, in an 8 -h shift, a 10 - by $10-f t$ heading can be advanced 30 ft , yielding 220 st of cuttings.

## CUTTER FORCE AND POWER CALCULATIONS

Using the experimental data, the forces acting on the cutter and the power required to excavate at a rate of 220 st per shift can also be calculated.

1. The laboratory studies showed that the efficiency factor $N_{R}$ for ripper cutting ranges up to 6.2. Assuming an efficiency factor of 6 and a rock compressive strength of $18,000 \mathrm{lbf} / \mathrm{in}^{2}$, the specific energy is calculated (as explained in the section, "SPECIFIC ENERGY OF CUTTING") to be $18,000 / 6$, or $3,000 \mathrm{in} \cdot 1 \mathrm{bf} / \mathrm{in}^{3}$.
2. The cutter force is considered to be proportional to the area of the bit in contact with the rock times 3,000 lbf/in ${ }^{2}$. For a 12 -in-wide bit cutting 5 in deep, the area is $60 \mathrm{in}^{2} \times 3,000$ lbf/in ${ }^{2}$, which yields an average cutter
force of $180,000 \mathrm{lbf}$. The peak cutting force is 2.5 times the average cutting force, or $450,000 \mathrm{lbf}$.
3. The energy and horsepower requirements are computed by the total energy consumed per cut as follows: The volume cut per pass is 5 in deep by 12 by 120 in, or $7,200 \mathrm{in}^{3}$. Since the energy per unit volume is $3,000 \mathrm{lbf} \cdot \mathrm{in}_{\mathrm{n}} / \mathrm{in}^{3}$, the total energy consumed per cut is $1,800,000 \mathrm{ft} \cdot 1 \mathrm{bf}$ 。
4. The average horsepower requirement for the ripper miner is the total energy used divided by the time of the swing, which yields 131 hp .

## SUMMARY AND CONCLUSIONS

The specific conclusions that can be made based on the experimental data are as follows:

1. The ripper cutting method has demonstrated the ability to excavate rectangular-shaped openings of 6 by 6 ft . This is considered full-scale cutting. The method is capable of excavating larger size openings, but the maximum size has not yet been defined.
2. The ripper cutting method is able to cut full scale in $9,200-1 \mathrm{bf} / \mathrm{in}^{2}$ concrete, and smaller scale tests have shown the method capable of cutting rocks up to $27,0001 \mathrm{bf} / \mathrm{in}^{2}$. The hardness limit of the method has not yet been defined.
3. The best energy efficiency achieved by the method, as measured by the rock number $N_{R}$, is 6.2 , which compares to 4 to 6 for tunnel boring machines and 8 for roadheaders. This efficiency was achieved with the $10^{\circ}$ rake angle bit, cutting 9 in wide and 1.5 in deep.
4. The energy efficiency of the cutting process improved steadily as the cross-sectional area of the cut increased. Therefore, to achieve high efficiency cutting, the bit should make the largest cut possible within the limitations of the machine.
5. Both the $0^{\circ}$ and $10^{\circ}$ rake angle bits performed equally well in terms of energy efficiency, and there is no basis for choosing between them. It is known from other tests, however, that rake angle has a significant effect on efficiency, so
the selection of the optimum bit geometry remains a trial-and-error process for each rock type.

The cutting experiments described here have shown that the ripper cutting method can successfully cut the large, rectangular-shaped excavations required by the mining industry: Previous experiments showed that the method is capable of cutting a wide range of rocks ranging in strength from 5,000 to $27,000 \mathrm{lbf} / \mathrm{in}^{2}$, giving it a wide field of application. In addition, the ripper cutting method possesses certain other advantages which make it very useful for mining. These advantages are as follows: (1) The sim. ple, inexpensive drag cutters used (with no moving parts) would yield the lowest cutter cost per ton mined, compared to any type of rolling cutter; (2) the method has the potential to achieve high production rates which can be varied by simply adjusting the width and depth of the cut; (3) ripper cutting is not sersitive to geologic conditions such as mixed face, high water inflows, or blocky ground; and (4) cutter changes can be made quickly and easily by one operator when necessitated by wear or to be better match formation cutting characteristics.

Because of the success of these experiments, the ripper cutting method will continue to undergo further large-scale 1aboratory testing. The purpose of these tests will be to generate accurate cost data that will permit a realistic
economic feasibility analysis to be performed. A patent for a universal mining machine based on the ripper cutting
method, has been granted to the U.S. Government (7).

## REFERENCES

1. Gaye, F. Efficient Excavation With Particular Reference to Cutting Head Design of Hard Rock Machines. Tunnels and Tunnelling, v. 4, 1972, No. 1 (pt. 1), pp. 39-48; No. 2 (pt. 2), pp. 135-143.
2. Hignett, H. J., R. A. Snowdon, and J. Temporal. Rock Cutting Trials and Lower Chalk. Paper in Energy Resources and Excavation Technology, Proceedings 18th U.S. Symposium on Rock Mechanics (CO School of Mines, Golden, CO, June 22-24, 1977). CO School of Mines Press, 1977, 34 pp.
3. Hughes, H. M. Some Aspects of Rock Machining. Int. J. of Rock Mech. Min. Sci., v. 9, No. 2, 1972, pp. 205-211.
4. Larson, D. A., and R. C. Olson. Design Consideration of Mechanical

Fragmentation Systems for Entry Development in Oil Shale. Paper in Proceedings of 10 th Oil Shale Symposium (CO School of Mines, Golden, CO, Apr. 21-22, 1977). C0 School of Mines Press, 1977, pp. 99-107.
5. Morre11, R. J., D. A. Larson, and D. E. Swanson. Large-Scale Drag Cutter Experiments in Hard Rock. BuMines RI 9003, 1986, 18 pp .
6. Morrell, R. J., and R. J. Wilson. Toward the Development of a Hard Rock Mining Machine--Drag Cutter Experiments in Hard Abrasive Rocks. BuMines RI 8784, 1983, 19 pp .
7. Morrell, R. J., and D. A. Larson. Universal Ripper Miner. U.S. Pat. 4,501,448, Feb. 26, 1985.

APPENDIX.--RAW DATA

| Cut No. | $\begin{aligned} & \text { Rake } \\ & \text { angle, } \end{aligned}$ | Advance distance, in | Cut |  |  |  | Force, 1 bf |  |  |  |  |  | $\begin{gathered} \text { Cut } \\ \text { vol. }, \\ \text { in }^{3} \end{gathered}$ | $\begin{aligned} & \text { Total. } \\ & \text { energy, } \\ & \text { lbf•in } \end{aligned}$ | Specific energy,$\frac{i n \cdot 1 b f}{i n^{3}}$ | $\begin{gathered} \text { Av cut } \\ \text { pressure, } \\ \text { lbf/in }{ }^{3} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | Depth, in | $\begin{gathered} \text { Width, } \\ \text { in } \end{gathered}$ | $\begin{aligned} & \text { Length, } \\ & \text { in } \end{aligned}$ | Time, s | Mid cut | $\begin{array}{r} \text { Av } \\ \text { cut } \end{array}$ | Peak $\begin{aligned} & \text { Perce, } \\ & \text { cut }\end{aligned}$ | Mid <br> normal | $\begin{gathered} \text { Av } \\ \text { normal } \end{gathered}$ | Peak normal |  |  |  |  |
|  | 0 | 11.75 | 1.50 | 4.00 | 60.7 | 19.8 | 13,326 | 12,945 | NA | 7,800 | 8,500 | NA | 304 | 785,761 | in ${ }^{\text {a }}$, 585 | 1,820 |
| 2... | 0 | 11.75 | 1.50 | 4.00 | 60.5 | 18.4 | 14,311 | 11.096 | NA | 11,400 | 12,500 | NA | 302 | 671,308 | 2,585 | 1,820 1,560 |
| 4.. | 0 | 11.75 | 1.50 | 4.00 | 60.2 | 18.0 | 14,012 | 13,737 | NA | 14,200 | 13,800 | NA | 299 | 826,967 | 2,766 | 1,560 1,932 |
|  | 0 | 11.75 | 1.50 | 4.00 | 59.7 | 18.0 | 10,317 | 12,734 | NA | 13,000 | 11,500 | NA | 296 | 760,220 | 2,568 | 1,791 |
|  | 0 | 13.25 | 1.50 | 2.00 9.00 | 59.3 65.1 | 17.4 28.4 | 11.509 23,241 | 9,300 | NA | 8,500 | 8,200 | NA | 146 | 551,490 | 3,777 | 5,231 |
| 7. | 0 | 13.25 | 1.50 | 4.00 | 65.1 64.9 | 28.4 20.4 | 23,241 15,669 | 16,05 | 42,023 | 17,500 | 17,000 | 31,500 | 755 | 1,045,115 | 1,384 | 446 |
| 8. | 0 | 13.25 | 1.50 | 4.00 | 64.7 | 19.5 | 15,669 10,442 | 11,930 8,325 | 33,462 29,308 | 7,700 | 8,000 | 16,200 | 303 | 774,257 | 2,555 | 1,678 |
| 9.. | 0 | 13.25 | 1.50 | 4.00 | 64.5 | 21.1 | 10,442 | 11,325 11,879 | 29,308 | 14,000 | -9,300 | 14, 300 | 318 | 538,627 | 1,694 | 1,171 |
| 10. | 0 | 13.25 | 1.50 | 4.00 | 64.1 | 20.5 | 11,820 | 10,560 | 37, 502 | 14,500 9,500 | 12,200 10,500 | 20, 500 17,000 | 316 314 | 766,195 676,896 | 2,425 | 1,670 |
| 11. | 0 | 13.25 | 1.50 | 4.00 | 63.7 | 19.7 |  |  |  |  |  |  |  |  |  |  |
| 12. | 0 | 13.25 | 1.50 | 4.00 | 63.2 | 17.3 | 12,805 |  |  |  | 12,800 | 28,500 | 310 | 857,274 | 2,765 | 1,893 |
| 13. | 0 | 13.25 | '. 50 | 2.50 | 62.8 | 17.3 | 17,138 | 13,767 |  |  | 13,000 | 22,000 | 306 | 718,710 | 2,349 | 1,599 |
| 14. | 10 | 13.25 | 1.50 | 3.75 | 64.9 | 21.0 | 9,258 | 10,032 | 27,962 | 13,500 | 18,500 | 21,000 | 188 | 864,568 | 4,599 | 4,956 |
| 15. | 10 | 13.25 | 1.50 | 4.25 | 64.7 | 20.3 | 12,953 | 11,523 | 34,774 | 10,400 | 13,000 | 24,900 | 303 | 651,077 | 2,149 | 1,605 |
| 16. | 10 | 13.25 | 1.50 | 4.00 | 64.5 | 19.8 | 11,870 | 9,628 | 33,542 | 15,500 |  | 28,000 | 318 | 745,538 | 2,344 | 1,435 |
| 17.. | 10 | 13.25 | 1.50 | 4.00 | 64.1 | 20.7 | 12,400 | 11,925 | 29,706 | 18,500 | 12,000 | 27,000 26,500 | 316 314 | 621,006 | 1,965 | 1,354 |
| 18. | 10 | 13.25 | 1.50 | 4.00 | 63.7 | 18.9 | 10,442 | 10,913 | 36,862 | 14,000 | 12,500 | 23,500 | $\begin{array}{r}314 \\ 310 \\ \hline\end{array}$ | 764,393 | 2,434 | 1,677 |
| 19. | 10 | 13.25 | 1.50 | 4.00 | 63.2 | 18.9 | 15,366 | 15,980 | 38,253 | 19,000 | 17,500 | 31,000 | 310 | 695,158 $1,009,936$ | 2,242 3,300 | 1,535 2,247 |
| 20.. | 10 | 14.75 | 1.50 | 9.00 | 68.9 | 22.5 | 17,335 | 16,873 | 42,288 | 28,000 | 23,500 | 41,000 | 789 | 1,162,549 | 1.473 | 469 |
| 21.. | 10 | 14.75 | 1.50 | 2.00 | 68:8 | 21.5 | 9,944 | 9,522 | 14,340 | 15,000 | 13,000 | 15:500 | 147 | 655,114 | 4,457 |  |
| 22... | 10 | 14.75 | 1.50 | 2.00 | 68.7 | 21.6 | 10,835 | 9,060 | 17,640 | 17,640 | 9,500 | 12,500 | 166 | 622,422 | 3,750 | $\begin{aligned} & 5,356 \\ & 5,096 \end{aligned}$ |
| 23... | 10 | 14.75 | 1.50 | 2.00 | 68.7 | 21.6 | 11,426 | 10,223 | 19,486 | 10,000 | 7,000 | 11,000 | 165 | 702,320 | 4,256 | 5,096 5,750 |
| 24. | 10 | 14.75 | 1.50 | 2.00 | 68.4 | 21.3 | 9.130 | 8,918 | 17,159 | 9,000 | 8,000 | 9,800 | 165 | 609,991 | 4,256 |  |
| 25. | 10 | 14.75 | 1.50 | 2.00 | 68.3 | 21.1 | 11,225 | 11,449 | 19,409 | 10,500 | 9,000 | 11,500 | 164 | 781,967 | 4,768 | 6,440 |
| 26... | 10 | 14.75 | 1.50 | 2.00 | 68.1 | 21.5 | 10,934 | 10,829 | 23,861 | 12,600 | 10,000 | 12,400 | 163 | 737,455 | 4,524 | 6,091 |
| 27. | 10 | 14.75 | 1.50 | 2.00 | 67.9 | 21.5 | 10,048 | 8,547 | 20,681 | 12,300 | 7,500 | 12,800 | 163 | 580,341 | 3,560 | 4,808 |
| 28. | 10 | 14.75 | 1.50 | 2.00 | 67.7 | 21.2 | 11,377 | 9,649 | 21,317 | 10,000 | 8,600 | 11,500 | 162 | 653,237 | 4,032 | $\begin{aligned} & 4,808 \\ & 5,428 \end{aligned}$ |
| 29. | $\vdots 0$ | 14.75 | 1.50 | 2.00 | 67.5 | 20.7 | 11,081 | 11,004 | 17,143 | 10,000 | 9,000 | 10,700 | 161 | 742,770 | 4,613 | 6,190 |
| 30. | 10 | 14.75 | 1.50 | 2.00 | 67.2 | 20.5 | 9,555 | 9,620 | 15,095 | 14,500 | 13,000 | 14,500 | 160 | 646,464 | 4,040 |  |
| 31. | 10 | 14.75 | 1.50 | 2.00 | 66.9 | 19.8 | 10,589 | 9,2.18 | 15,314 | 9,000 | 7,000 | 9,000 | 159 | 616,684 | 3,879 | $\begin{aligned} & 5,411 \\ & 5,185 \end{aligned}$ |
| 22.. | 10 | 14.75 | 1.50 | 2.00 | 66.7 | 19.6 | 13,790 | 11,796 | 21,873 | 22,000 | 17,200 | 22,500 | 157 | 786,793 | 5,011 | 5,185 6,635 |
| 33. | 0 | 14.75 | 1.50 | 2.00 | 68.8 | 22.2 | 9,949 | 8,199 | 18,373 | 8,500 | 6.500 | 10,000 | 147 | 564,091 | 3,837 | 6,635 |
| 34. | 0 | 14.75 | 1.50 | 2.00 | 68.7 | 21.6 | 11,180 | 8,103 | 20,365 | 13,500 | 9,800 | 13,000 | 166 | 556,676 | 3,837 | 4,612 |
| 35... | 0 | 14.75 | 1.50 | 2.00 | 68.7 | 21.4 | 9,703 | 8,733 | 19,369 | 9,000 | 6,500 | 13,000 | 166 | 599,957 | 3,614 | 4,558 4,912 |
| 36. | 0 | 14.75 | 1.50 | 2.00 | 68.5 | 21.4 | 9,949 | 8,929 | 29,209 | 8,000 | 7,500 | 10,800 | 165 | 611,636 | 3,614 3,707 | 4,912 5,023 |
| 37. | 0 | 14.75 | 1.50 | 2.00 | 68.4 | 21.6 | 8,964 | 8,852 | 19,568 | 8,000 | 7,000 | 8,800 | 165 | 605,477 | 3,670 | $\begin{aligned} & 5,023 \\ & 4,979 \end{aligned}$ |
| 38. | 0 | 14.75 | 1.50 | 2.00 | 68.3 | 21.5 | 9,949 | 9,901 | 24,915 | 9,000 | 8,600 | 13,000 | 164 | 676,238 | - 4,123 | $\begin{aligned} & 4,979 \\ & 5,569 \end{aligned}$ |
| 39... | 0 | 14.75 | 1.50 | 2.00 | 68.1 | 21.1 | 9,415 | 9,453 | 21,933 | 7,700 | 7,000 | 9,400 | 163 | 643,749 | 4,949 | 5,569 5,317 |

RAW DATA--Cont inued

| Cut <br> No. | Rake angle, | Advance distance, in | Cut |  |  |  | Force, 1 bf |  |  |  |  |  | Cut vol., $i^{3}{ }^{3}$ | $\begin{aligned} & \text { Total } \\ & \text { energy, } \\ & \text { 1.bf.in } \end{aligned}$ | $\begin{aligned} & \text { Specific } \\ & \text { energy, } \\ & \frac{\text { in } \cdot 1 b \mathrm{f}}{\mathrm{in}^{3}} \end{aligned}$ | Av cut pressure, 1bf. /in |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | Depth, | $\begin{gathered} \text { Width, } \\ \text { in } \end{gathered}$ | $\begin{aligned} & \text { Length, } \\ & \text { in } \end{aligned}$ | Time, s | $\begin{aligned} & \text { Mid } \\ & \text { cut } \end{aligned}$ | $\begin{aligned} & \text { Av } \\ & \text { cut } \end{aligned}$ | Peak cut | Miત normal | Av normal. | Peak norinal |  |  |  |  |
| 40.. | 0 | 14.75 | 1.50 | 2.00 | 67.9 | 21.2 | 7,979 | 8,518 | 21,754 | 13,000 | 10,000 | 13,400 | 163 | 578,372 | 3,548 | 4,791 |
| 41 | 0 | 14.75 | 1.50 | 2.00 | 67.7 | 21.2 | 9,703 | 8,511 | 17,977 | 8,400 | 8,600 | 9,000 | 162 | 576,195 | 3,557 | 4,787 |
| 42. | 0 | 14.75 | 1.50 | 2.00 | 67.5 | 20.2 | 7,979 | 8,706 | 22,152 | 11,500 | 10,500 | 14,000 | 161 | 587,655 | 3,650 | 4,897 |
| 4 | 0 | 14.75 | 1.50 | 2.00 | 67.2 | 20.6 | 9,456 | 7,902 | 19,568 | 7,200 | 8,400 | 9,000 | 160 | 531,014 | 3,319 | 4,445 |
| 44 | 0 | 14.75 | 1. 50 | 2.00 | 66.9 | 21.0 | 14,381 | 12,394 | 25,199 | 12,800 | 11, 100 | 15,300 | 159 | 829,158 | 5,215 | 6,972 |
| 45. | 0 | 16.25 | 1.50 | 9.00 | 72.6 | 22.5 | 20,290 | 19,139 | 40,002 | 23,000 | 24,000 | 44,000 | 820 | 1,389,491 | 1,695 | 532 |
| 46. | 0 | 16.25 | 1.50 | 6.00 | 72.3 | 21.7 | 16,351 | 15,659 | 41,732 | 24,000 | 20,000 | 39,000 | 494 | 1,132,146 | 2,292 | 979 |
|  | 0 | 16.25 | 1. 50 | 6.00 | 71.9 | 21.8 | 19,797 | 15,673 | 37,361 | 23,500 | 20,500 | 39,000 | 511 | 1,126,888 | 2,205 | 980 |
| 48. | 0 | 16.25 | 1.50 | 6.00 | 71.3 | 20.7 | 16,351 | 15,759 | 40,837 | 14,500 | 16,200 | 35,000 | 505 | 1,123,617 | 2,225 | 985 |
| 49. | 0 | 16.25 | 1.50 | 6.00 | 70.5 | 20.4 | 16,351 | 13,401 | 38,452 | 19,000 | 19,500 | 35,500 | 496 | 944,770 | 1,905 | 838 |
| 50. | 0 | 16.25 | 1.50 | 4.00 | 69.6 | 18.0 | 25,706 | 21,711 | 47,397 | 23,500 | 23,000 | 42,000 | 314 | 1,511,086 | 4,812 | 3,053 |
| 51. | 0 | 16.25 | 1. 50 | 6.00 | 72.3 | 20.4 | 12,904 | 14,720 | 34,277 | 20,000 | 22,500 | 37,000 | 494 | 1,064,256 | 2,154 | 920 |
| 52. | 10 | 16.25 | 1.50 | 6.00 | 71.9 | 19.5 | 15,366 | 13,895 | 33,085 | 21,500 | 18,500 | 41,000 | 511 | -999,050 | 1,955 | 868 |
| 53. | 10 | 16.25 | 1.50 | 6.00 | 71.3 | 19.8 | 13,642 | 13,126 | 31,296 | 23,000 | 20,000 | 30,500 | 505 | 935,884 | 1,853 | 820 |
| 54. | 10 | 16.25 | 1.50 | 6.00 | 70.5 | 19.8 | 19,305 | 17,226 | 35,470 | 23, 300 | 21,000 | 32,000 | 496 | 1,214,433 | 2,448 | 1,077 |
| 55. | 10 | 18.25 | 2.00 | 9.00 | 77.4 | 25.6 | 30,140 | 26,506 | 58,727 | 47,500 | 28,000 | 57,000 | 1,136 | 2,051,564 | 1,806 | 1,309 |
| 56 | 10 | 18.25 | 2.00 | 4.00 | 77.2 | 20.4 | 16,844 | :7,034 | 37,259 | 22,500 | 21,000 | 44,000 | 441 | 1,315,025 | 2,982 | 4,259 |
| 57 | 10 | 18.25 | 2.00 | 4.00 | 76.9 | 25.6 | 15,859 | 18,914 | 51,571 | 22,000 | 22,500 | 42,000 | 472 | 1,454,486 | 3,082 | 4,729 |
| 58. | 10 | 18.25 | 2.00 | 4.00 | 76.6 | 21.0 | 22,261 | 16,837 | 39,645 | 22,300 | 21,500 | 32,000 | 469 | 1,289,714 | 2,750 | 4,209 |
| 59.. | 10 | 18.25 | 2.00 | 4.00 | 76.2 | 19.2 | 14,874 | 14,204 | 36,066 | 15,500 | 17,000 | 29,000 | 465 | 1,082,345 | 2,328 | 3,551 |
| 60. | 10 | 18.25 | 2.00 | 4.00 | 75.7 | 19.8 | 12,904 | 13,893 | 38,452 | 18,500 | 16,000 | 28,500 | 460 | 1,051,700 | 2,286 | 3,473 |
| 61 | 10 | 18.25 | 2.00 | 4.00 | 75.1 | 20.1 | 13,151 | 12,740 | 36,066 | 14,000 | 15,000 | 24,000 | 454 | 956,774 | 2,107 | 3,185 |
| 62. | 10 | 18.25 | 2.00 | 4.00 | 74.9 | 20.1 | 23,245 | 18,412 | 40,241 | 23,000 | 20,000 | 31,500 | 447 | 1,379,059 | 3,085 | 4,603 |
| 63. | 0 | 18.25 | 2.00 | 4.00 | 77.2 | 21.0 | 15,858 | 14,616 | 46,204 | 20,500 | 17,500 | 37,500 | 441 | 1,128,355 | 2,559 | 3,554 |
| 64... | 0 | 18.25 | 2.00 | 4.00 | 76.9 | 19.8 | 19,305 | 16,891 | 47,397 | 21,000 | 21, 800 | 40,500 | 472 | 1,298,918 | 2,752 | 4,223 |
| 65... | 0 | 18.25 | 2.00 | 4.00 | 76.6 | 20.8 | 20,290 | 17,800 | 50,379 | 20,000 | 17,500 | 37,500 | 469 | 1,363,480 | 2,907 | 4,450 |
| 66. | 0 | 18.25 | 2.00 | 4.00 | 76.2 | 20.1 | 18,566 | 18,057 | 52, 168 | 23,500 | 24,200 | 39,000 | 465 | 1,375,943 | 2,959 | 4,514 |
| 67. | 0 | 18.25 | 2.00 | 4.00 | 75.1 | 21.0 | 22,260 | 16,490 | 46,204 | 23,000 | 21,500 | 38,500 | 460 | 1,238,399 | 2,692 | 4,123 |
| 68... | 0 | 18.25 | 2.00 | 4.00 | 74.4 | 21.0 | 22,752 | 20,449 | 52,764 | 22,500 | 21,000 | 36,500 | 454 | 1,521,405 | 3,351 | 5,112 |
| 69. | 0 | 19.25 | 1.00 | 9.00 | 79.7 | 19.2 | 19,305 | 16,691 | 17,856 | 18,500 | 16,500 | 29,000 | 584 | 1,330,273 | 2,278 | 206 |
| 70. | 0 | 19.25 | 1.00 | 4.00 | 79.5 | 20.7 | 11,919 | 11,294 | 30,699 | 14,500 | 14,500 | 24,500 | 225 | 897,873 | 3,991 | 706 |
| 71. | 0 | 19.25 | 1.00 | 4.00 | 79.2 | 20.8 | 14,381 | 12,055 | 30,302 | 17,500 | 15,500 | 25,000 | 242 | 954,756 | 3,945 | 753 |
| 72. | 0 | 19.25 | 1.00 | 4.00 | 78.9 | 20.8 | 14,381 | 11,549 | 31,494 | 19,500 | 16,000 | 30,500 | 240 | 911,216 | 3,797 | 722 |
| 73. | 0 | 19.25 | 1.00 | 4.00 | 78.5 | 20.4 | 11,426 | 11,919. | 27,519 | 12,500 | 15,000 | 23,000 | 238 | 935,641 | 3,931 | 745 |
| 74. | 0 | 19.25 | 1.00 | 4.00 | 78.0 | 18.0 | 17,089 | 11,169 | 26,525 | 16,000 | 15,500 | 24,500 | 236 | 871,182 | 3,691 | 698 |
| 75. | 0 | 19.25 | 1.00 | 4.00 | 77.4 | 19.6 | 8,472 | 9,216 | 23,543 | 12,000 | 12,500 | 21,500 | 233 | 713,318 | 3,061 | 576 |
| 76. | 0 | 19.25 | 1.00 | 2.00 | 76.6 | 20.8 | 18.813 | 16,548 | 33,880 | 14,000 | 20,500 | 32,500 | 115 | 1,267,577 | 11,022 | 4,137 |
| 77. | 10 | 19.25 | 1.00 | 4.00 | 79.5 | 20.7 | 14,873 | 11, 407 | 26,823 | 20,500 | 17,500 | 26,500 | 225 | 906,856 | 4,030 | 713 |
| 78. | 10 | 19.25 | 1.00 | 4.00 | 79.2 | 21.0 | 9,949 | 11,132 | 28, 115 | 15,000 | 17,000 | 28,500 | 242 | 881,654 | 3,643 | 696 |
| 79.. | 10 | 19.25 | 1.00 | 4.00 | 78.9 | 21.6 | 10,934 | 9,846 | 27,718 | 20,000 | 17,500 | 27,500 | 240 | 776,849 | 3,237 | 615 |
| 80. | 10 | 19.25 | 1.00 | 4.00 | 78.5 | 21.7 | 11,919 | 10,747 | 26, 525 | 27,500 | 22,500 | 32,000 | 238 | 843,639 | 3,545 | 672 |
| 81. | 10 | 19.25 | 1.00 | 4.00 | 78.0 | 21.9 | 10,442 | 10,567 | 25,332 | 18,500 | 15,500 | 24,000 | 236 | 824,226 | 3,492 | б60 |
| 82... | 10 | 19.25 | 1.00 | 4.00 | 77.4 | 22.2 | 16,843 | 15,171 | 32,290 | 25,500 | 22,000 | 34,000 | 233 | 1,174,235 | 5,040 | 948 |


[^0]:    ${ }^{1}$ Mining engineer.
    ${ }^{2}$ Supervisory mining engineer.
    Twin Cities Research Center, Bureau of Mines, Minneapolis, MN.

[^1]:    $3_{\text {Underlined numbers in parentheses re- }}$ fer to items in the list of references preceding the appendix.

[^2]:    l"Mesh" indicates Tyler mesh.

